

# **Faculty of Mechanical Engineering**

# CFD ANALYSIS OF INTAKE FLOW IN THE L-HEAD ENGINE

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# CFD ANALYSIS OF INTAKE FLOW IN THE L-HEAD ENGINE

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A thesis submitted
In fulfilment of the requirement for the degree of Master of Science in Mechanical Engineering

**Faculty of Mechanical Engineering** 

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### **DECLARATION**

I declare that this thesis entitle "CFD Analysis of intake flow in the L-head engine" is the result of my own research except as cited in the references. The thesis has not been accepted for any degree and is not concurrently submitted in candidature of any other degree.

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# **APPROVAL**

I hereby declare that I have read this thesis and in my opinion this thesis is sufficient in
terms of scope and quality for the award of Master of Science in Mechanical Engineering.

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Date	ː

# **DEDICATION**

I dedicate this thesis to my family for nursing me with affections and love and their dedicated partnership for success in my life.

#### **ABSTRACT**

The CNG fuel engine usually had lower performance comparing to gasoline fuel. A small single cylinder engine (L-head engine) is used in Green Technology Vehicle laboratory (GTeV) lab, which is low cost and commonly available in the market. It is difficult to experimentally to track the particle in 3D cases. Therefore, commercial numerical simulation is used. This thesis analysed the behaviour of the flow inside L-head combustion chamber for in-cylinder engine without the combustion with three different models were simulated. The objectives are to develop numerical model, to investigate the flow without combustion, to analyse flow with CNG and gasoline and to perform validation on the pressure between experiment and simulation. There are three types of simulation; steady, Port-flow and Cold-flow. The engine parameters and valve lift are measured, and the engine head were scan using ezScan 4.5 software. All of the simulations are simulated using ANSYS software. Only intake stroke is simulated for steady simulation with different crank angle and engine speed. Port-flow simulation is only simulated at intake stroke with introduction of CNG and gasoline as a fuel and three different valve lift. Cold-flow simulated a full cycle engine without combustion process. The Steady simulation is dealing with the static domain. There are only combustion chambers and piston volume involved in the steady simulation. The air inlet velocity was calculated using the standard engine formula for different piston position. The second simulation, Port-flow simulation also deals with the static geometry domain. Inplenum and outplenum was added by the presence of both intake and exhaust valve for the Port flow simulation domain. Three different valve lift was chosen. Gasoline and CNG were used as fuel, which enters the domain through the fuel intake. The last simulation is called Coldflow where the geometry is moving according to the crank angle. The intake valve and exhaust valve are moving according to the measured valve profile. Meanwhile the piston movement was generated according to the crank angle of the engine. The result of steady flow simulation shows the velocity is high when the piston position is at 45° and engine speed of 4500 rpm. The result of Port-flow simulation shows the mass flow rate and velocity across the domain increase as the valve lift increase. The pressure difference between the intake port and combustion chamber decrease as valve lift increase. The swirl ratio decreases as going down the cylinder. The Cold-flow result shows the turbulence kinetic energy, swirl, tumble, and cross-tumble ratio inside the combustion chamber increase in the middle of the intake stroke. The temperature inside combustion chamber is increasing as the piston reaches TDC due to compression process. The result of Cold-flow simulation is validated by experiment without combustion with 22.73% of percentage difference at peak pressure. The combustion chamber head has been scanned and imported to ANSYS software. The velocity is highest when the piston located at the middle of the stroke and lowest then the piston approaching TDC and BDC. The flow pattern of gasoline and CNG has no significant change. The pressure for experiment and Cold-flow simulation is validated through its pressure pattern.

#### ABSTRAK

Enjin CNG mempunyai prestasi yang rendah berbanding petrol. Sebuah enjin kecil satu silinder (enjin kepala-L) digunakan, di dalam Makmal Kenderaan Hijau (GTeV) yang mempunyai kos yang rendah dan biasa terdapat di pasaran. Adalah sukar untuk menjejak zarah dalam bentuk 3D. Oleh itu, simulasi berangka komersial digunakan. Tesis ini menganalisis kelakuan aliran didalam kebuk pembakaran enjin berkepala L tanpa pembakaran dengan tiga model yang berlainan. Obejktif adalah untuk membangunkan model berangka, untuk menganalisis aliran dengan CNG dan petrol dan melaksanakan pengesahan pada tekanan diantara eksperimen dan simulasi. Terdapat tiga jenis simulasi; kekal, aliran-rongga, dan aliran-sejuk. Parameter enjin dan valve diukur dan kepala enjin telah diimbas menggunakan perisian ezScan 4.5. Semua simulasi hanya menggunakan perisian ANSYS. Hanya lejang pengambilan diambil kira untuk simulasi kekal. Simulasi aliran-rongga disimulasi pada lejang masukan dengan pengenalan CNG dan petrol sebagai bahan api dengan tiga ketinggian injap yang berbeza. Kitaran penuh digunakan untuk simulasi aliran-sejuk tanpa proses pembakaran. Simulasi kekal adalah berkaitan dengan domain statik. Hanya isipadu kebuk pembakaran dan omboh diambik. Halaju udara masuk dikira menggunakan formula standard yang digunakan untuk pengiraan enjin. Simulasi kedua aliran-rongga yang juga menggunakan domain geometri static. Inplanum dan outplenum ditambah dengan kehadiran kedua-dua injap pengambilan dan ekzos untuk domain aliran-port. Terdapat juga tiga perbezaan kenaikan injap. Petrol dan juga CNG digunakan sebagai bahan api dimana ia masuk ke dalam domain melalui permukaan kemasukan bahan api. Simulasi yang terakhir dipanggil aliran-sejuk dimana geometri akan bergerak mengikut pada sudut engkol. Injap kemasukan dan injap ekzos bergerak mengikut profil injap yang diukur. Sementara itu, pergerakan omboh dihasilkan mengikut sudut engkol engin. Keputusan untuk simulasi aliran kekal menunjukkan halaju tinggi apabila omboh berada pada 45° dan enjin pada 4500 rpm untuk kedua-dua sudut. Keputusan aliran- rongga menunjukkan seluruh kadar aliran jisim dan halaju di dalam domain meningkat apabila injap menaik. Perbezaan tekanan di antara rongga kemasukan dan kebuk pembakaran menurun apabila tinggi injap menaik. Penurunan nisbah pusaran berlaku apabula aliran mengalir menjauhi kedalam silinder. Keputusan aliran-sejuk membincangkan berkenaan pergolakan tenaga kinetic, nisbah pusaran, nisbah tumble, dan nisbah bertentangan tumble di dalam kebuk pembakaran menaik pada pertengahan proses lejang masuk. Suhu didalam kebuk pembakaran menaik apabila omboh menghampiri kedudukan TDC pada lejang mampatan. Hasil simulatsi aliran-sejuk disahkan dengan eksperimen tanpa pembakaran dengan 22.73% perbezaan peratus pada tekanan memuncak. Kepala kebuk pembakaran telah diimbas dan diimport ke perisian ANSYS. Halaju paling tinggi apabila omboh terletak di tengah-tengah lejang dan kembali rendah bila menghampiri TDC dan BDC. Corak aliran untuk petrol dan CNG tidak mempunyai perubahan ketara. Tekanan eksperimen dan simulate aliran-sejuk disahkan dengan corak tekanan yang terhasil.

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# **TABLE OF CONTENTS**

		PAGE
DE	CCLARATION	
AP	PROVAL	
DE	EDICATION	
ABSTRACT ABSTRAK		i
		ii
	CKNOWLEDGEMENT	iii
	ABLE OF CONTENTS	iv
LIST OF TABLES LIST OF FIGURES		vi
		vii
	ST OF ABBREVIATION AND SYMBOLS	
	ST OF APPENDICES	X
		XV
LI	ST OF PUBLICATIONS	xvi
CH	HAPTER	
1.	INTRODUCTION	1
	1.1 Overview	1
	1.2 Problem Statement	2
	1.3 Objective	3
	1.4 Scope	3
	Scope	3
2.	LITERATURE REVIEW	5
	2.1 Theory of Four-Stroke Engine	5
	2.2 CNG as an Alternative Fuel	10
	2.3 Background of L-head engine	17
	2.4 Review of previous visualization method	20
	2.5 Introduction to CFD	30
	2.0 miliodadion to CFB	30
<b>3.</b>	RESEARCH METHODOLOGY	40
	3.1 Overview	40
	3.2 Simulation Flow Chart	41
	3.3 Experiment Set-Up	43
	3.4 Detail of L-Head	46
	3.4.1 Domain geometry	46
	3.4.2 Valve lift profile	52
	3.5 Meshing	55
	3.5.1 Meshing For Steady Simulation	55
	3.5.2 Meshing For Port-Flow	56
	3.5.3 Meshing For Cold-Flow	57
	3.6 Numerical Set-up	61
	3.6.1 Steady Simulation	61
	3.6.2 Port-Flow Simulation	62
	3.6.3 Cold-Flow Simulation	66
	5.5.5 Cold I low billidiation	00
4.	RESULT AND DISCUSSION	71
	4.1 Steady Flow Analysis	71
	4.2 Port-Flow Analysis	76

	4.3 Cold-Flow Analysis	95
5.	CONCLUSION AND RECOMMENDATION	108
	5.1 Overview	108
	5.2 Conclusions	108
	5.3 Recommendations	110
RE	FERENCES	111
AP	PENDIX A	122
AP	PENDIX B	126
AP	PENDIX C	130
AP	PENDIX D	136
AP	PENDIX E	138
AP	PENDIX F	139
AP	PENDIX G	141

# LIST OF TABLES

TABLE	TITLE	PAGE
2.1	Properties of gasoline and CNG (Maji, 2004)	11
2.2	Price comparison between the fuels in Malaysia on March 1,	13
	2015	
2.3	Compression ratio for various types of L-head combustion	20
	chamber (Mike, 2011)	
2.4	Comparison between CFD code by Novkovic et al.(2016)	24
2.5	Skewness ranges and cell quality	32
3.1	Specification of engine used	43
3.2	Initial condition and boundary condition for Steady simulation	62
3.3	Initial condition and boundary condition for Port-flow	64
	simulation	
3.4	Initial condition and boundary condition for Cold-flow	67
	simulation	
3.5	Dynamic zone in the Cold-flow simulation	68

# **LIST OF FIGURES**

FIGURE	TITLE	PAGE
2.1	Pressure versus crank angle for a full cycle	8
2.2	Oil demand by sector according to year OPEC (2014)	14
2.3	Greenhouse gaseous percentage	15
2.4	Outline for an ideal operation of CNG engine Bakar et al.	17
	(2012)	
2.5	Improvement of side valve combustion chamber. (Roger,	18
	2014)	
2.6	Various type of L-head engine with different compression	19
	ratio (Mike, 2011)	
2.7	Swirl and Tumble motion within combustion chamber	39
3.1	Flow chart of the research methodology	42
3.2	Illustration of the experiment set-up	44
3.3	High pressure sensor used in experiment	45
3.4	Location of pressure sensor	45
3.5	Scanning of engine head	46
3.6	Clearance volume of the in-cylinder engine and imported	47
	domain in ANSYS workbench	
3.7	Steady computational domain	48

3.8	Geometry is built according to desired angle	49
3.9	Port-flow domain	50
3.10	Cold-flow domain	52
3.11	Measuring of valve lift	53
3.12	Valve lift versus crank degree	54
3.13	Valve timing for full cycle	54
3.14	Cut-view of the clearance volume of the engine	56
3.15	Cut-view of Port-flow domain	57
3.16	Decomposition of the zones within the domain	58
3.17	Arrangement of the different zone and interfaces	59
3.18	Piston zone and interface between combustion chamber	60
3.19	Cut view Meshing of Cold-flow domain	60
4.1	Location of point where velocity magnitude monitored	72
4.2	Velocity magnitudes for various mesh size at chosen location	72
4.3	Reference plane of cutaway for steady simulation result	73
4.4	Flow viewed from Plane A-A at engine speed 4500 rpm	74
4.5	Flow viewed from plane B-B at 4500 rpm 18 ATDC	75
4.6	Flow viewed from plane B-B at 4500 rpm 45 ATDC	76
4.7	Locations of velocity monitored	77
4.8	Results for the port-flow study	77
4.9	Net mass imbalance according to valve lift through the	79
	domain	
4.10	Mass flow rate at the inplanum according to iteration	79
4.11	Mass flow rate at the inplanum based on valve lift distance	81

4.12	Velocity magnitude at plane C	82
4.13	Illustration of the plane location to display the velocity vector,	83
	pressure contour and velocity contour result	
4.14	Pressure and velocity contour	85
4.15	Swirl ratio at swirl plane 1	87
4.16	Swirl ratio at swirl plane 2	88
4.17	Swirl ratio at swirl plane 3	88
4.18	Velocity vector at the swirl plane at valve lift of 2.15	91
4.19	Velocity vector at the swirl plane at valve lift of 3.26	93
4.20	Velocity vector at the swirl plane at valve lift of 6.52	94
4.21	Pressure inside the combustion chamber for grid study	97
4.22	Motion of valves, and piston as a function on the crank angle	98
4.23	Volume inside the chamber as a function of time step	98
4.24	Iteration number as a function of time step	100
4.25	In-cylinder pressure as a function of crank angle	102
4.26	Temperature as a function of time step	103
4.27	Turbulence Kinetic Energy within the combustion chamber as	104
	a function of time step	
4.28	Swirl, Tumble, and Cross-tumble ratio inside the combustion	106
	chamber as a function of crank angle	
4.29	View of velocity vector at intake port at 50 degree	107
4.30	View of velocity vector at exhaust port at 240 degree	107

### LIST OF ABBREVIATIONS AND SYMBOLS

A/F Air Fuel

APEC Asia-Pacific Economic Cooperation

ATDC After Top Dead Centre

BDC Bottom Dead Centre

BTDC Before Top Dead Centre

CAD Computational-aided Design

CFD Computational Fluid Dynamic

CNG Compress Natural Gas

DAQ Data acquisition

DES Detached Eddy Simulation

DNS Direct Numerical Simulation

EVC Exhaust Valve Closed

EVM Eddy Viscosity Model

EVO Exhaust Valve Open

FDM Finite Difference Method

FEM Finite Element Method

FVM Finite Volume Method

GTeV Green Technology Vehicle

HP Horsepower

HWA hot wire anemometry

ICE Internal Combustion Engine

IEA International Energy Agency

IG Ignition Point

IVO Intake Valve Open

Ktoe Kilotonne of Oil Equivalent

LDA Laser Doppler Anemometry

LDV Laser Doppler Velocimetry

LES Large Eddy Simulation

LNG Liquefied Natural Gas

LPG Liquefied Petroleum Gas

NG Natural Gas

NGV Natural Gas Vehicle

OpenFOAM Open Field Operation And Manipulation

PIV Particle Image Velocimetry

PTV Particle Tracking Velocimetry

RANS Reynolds Average Navier-Stokes

RNG Re-Normalization Group

RPM Revolution Per Minute

RPS Revolution Per Second

RSM Reynolds Stress Model

SST Shear Stress Transport

TDC Top Dead Center

TKE Turbulence Kinetic Energy

UDF User Defined Function

3D Three-dimensional

Re Reynolds number

 $v_{avg}$  flow average velocity

V Flow velocity

 $D_h$  Characteristic length of geometry

 $D_{hs}$  Characteristic length of spherical geometry

D inner diameter of pipe

*S* stroke length

a crankshaft offset or radius of the crank shaft

 $\overline{U}p$  average piston speed

distance between the crank axis and the wrist pin axis

r length of the connecting rod

R ratio of the connecting rod length to crank is offset

 $U_p$  Instantaneous piston speed

 $r_c$  compression ratio

*v<sub>d</sub>* Volume Displacement

 $v_c$  Clearance volume

 $v_{cyl}$  Cylinder volume

B Bore

A Cross sectional area

Q Volume flow rate

 $k_2$  Theoretical value of compression ratio

 $k_1$  Simulation value of compression ratio

 $R_s$  Swirl ratio

 $R_t$  Tumble ratio

 $\omega_s$  Angular velocity

N Engine speed

*p* Piston position

y Length of rectangle duct

*n* Height of rectangle duct

W Diameter of circular area

j Radius of sphere

am Angular momentum per unit mass

 $\vec{r}$  Radial coordinate

 $\vec{v}$  Velocity

 $s\vec{a}$  Swirl axis

SN Swirl number

 $L_{sa}$  Magnitude of angular momentum respect to swirl axis

 $I_{sa}$  Moment of inertia of fluid mass about swirl axis

d Diameter

*u* Fluid velocity

 $Q_{air}$  Air flow rate

 $Q_{fuel}$  Fuel flow rate

ρ Density

*u* dynamic viscosity

 $\pi$  Pi (3.14159)

 $\theta$  crank angle

# LIST OF APPENDICES

APPENDICES	TITLE	PAGE
A	Engine geometric parameters	122
В	Calculation of Air Velocity	126
C	Mass Fraction and AFR for CNG and	130
С	gasoline	
D	Flow viewed from Plane A-A	136
E	Flow viewed from Plane B-B	138
F	Isosurface for Temperature in Cold-flow	139
F	Simulation	
G	Isosurface for TKE in Cold-flow Simulation	141

### LIST OF PUBLICATIONS

**Mohd Shafie, A. M.**, Musthafah, M. T., Ali, M. S., Bakar, R. A., Mohamed Arifin, Y., 2015. *Intake Analysis on Four-Stroke Engine Using CFD*. Journal of Engineering and Applied Sciences, 10(17), pp. 7799-7804

Ali, M. S., Musthafah, M. T., **Mohd Shafie, A. M.**, Bakar, R. A., 2014. *Simulation of Single Cylinder Engine Fuel with Alternative Fuel by Using Available Software*. International Review of Mechanical Engineering, 8(4), pp.798-802

Musthafah, M. T., Ali, M. S., Salim, M. A., Bakar, R. A., Fudhail, A. M., Hassan, M. Z., **Mohd Shafie, A. M.**, 2014. *Performance Analysis of Spark Ignition Engine Using Compressed Natural Gas (CNG) as Fuel*. Energy Procedia, 68, pp.335-362

#### **CHAPTER 1**

#### INTRODUCTION

#### 1.1 Overview

Improvement of the efficiency of the combustion engine is vital to diminish the global energy crisis and environmental impact of global warming. Fossil fuels such as gasoline and diesel are non-renewable energy, which keep decreasing as the demand of the energy increases over the year globally. International Energy Agency (IEA) (2014) found that growths of energy demand to the year of 2040 are increasing at about 37% with the average rate of growth of 1.1% per year. Meanwhile, the oil demand is rising at 14mb/d to achieve 104mb/d in 2040, with China as the largest oil-consuming country, despite the new policies all over the world promoting switching the fuel and enhancing the energy efficiency (IEA, 2014).

Devarajan (2015) stated that the reserve of fossil fuel around the world could only provide up to 40-50 years only as it is a finite amount. Due to this limitation of fossil fuel, research has to be more profound in order to utilize the highest efficiency of the system to reduce the consumption of the fuel. According to Deng and Liu (2013), saving one litre of gasoline can reduce the emission of carbon dioxide and carbon emission by 2.3 kg and 0.627 kg respectively.

In Malaysia, the primary energy supply is equivalent to 83 938 kilotons (ktoe) that comes from natural gas (46.0 %), oil (32.1 %), coal (18.9 %), hydro (2.9 %), and renewable energy (0.4 %). Here, the ktoe refers to an energy unit released with the amount of 1000 tons of oil burning, which is equivalent to 42 gigajoules. Total final energy

consumption in Malaysia in 2012 was 46 711 ktoe. Transport sector becomes the highest consumption, which is 36.8 % of the total final energy consumption (Malaysia Energy Statistics, 2014).

Environmental issues, such as global warming, have become the major factor to be focused. According to APEC (2013), carbon dioxide is estimated to increase by about 46% in year 2035 compared to the year 2010. Much legislation has been established to control the emission, leading to the reduction of emission, which started with Euro 1 standards in 1992 involving passenger car only.

The used of fuel for the experiment can be minimised by using a single cylinder engine rather than an actual commercial passenger car's engine. In Green Technology Vehicle (GTeV) laboratory, instead of using a four-cylinder engine for experiment at one time, the researcher can conduct other experiment using a single cylinder engine. L-head engine which is a small, low-cost and single-cylinder engine that commonly used in basic farm machinery. Furthermore, L-head engine and its spare parts is highly available in the market for a single cylinder engine segment. Due to this condition, the study of CFD simulation will be vital for the research.

#### 1.2 Problem Statement

(i) It is essentially important to study flow occurs in automotive engines. The flow criteria in engines such as swirl, values of turbulent kinetic energy are widely known to affect the engine performance. For example, high turbulent kinetic energy results in better air-fuel mixing, which consequently improves the engine performance. Experimental works are expensive and time consuming. Furthermore, it is difficult to experimentally track flow particle in three dimensional (3-D) cases. Hence, numerical simulations using commercial

softwares are the reliable alternative to experiments. There are limited numbers of numerical simulations work that focus on flows in a complete engine stroke. This study presents a fundamental numerical analysis of intake flow in a complete cycle engine. L-head shape engine is utilized as the computational domain. This type of engine head is different from the conventional engine.

(ii) L-head engine simulation can be hardly found due to its combustion chamber unique design. Therefore, the flow study have to be conducted in the GTeV laboratory.

### 1.3 Objectives

The objectives of this thesis have been set as follows:

- (i) To develop numerical model of a Robin EY-20-3 combustion chamber engine incorporate with valve lift profile.
- (ii) To investigate the flow inside a combustion chamber of a single cylinder Robin EY-20-3 engine without combustion process.
- (iii) To analyse the flow inside L-type in-cylinder Robin EY-20-3 engine using numerical analysis with Compressed Natural Gas (CNG) and gasoline.
- (iv) To perform validation on the built-up pressure obtained inside the Robin EY-20-3 engine between experiments and simulation.

### 1.4 Scopes

The job scopes of this research are as follows:-

(i) The engine parameters and valve lift profiles were measured and only Robin EY-20-3 engine head were scan using ezScan4.5 software. Only ANSYS Fluent are used for the numerical simulation.